

Rigid Body Pose Hybrid Control Using Dual Quaternions: Global Asymptotic Stabilization and Robustness

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A hybrid feedback control scheme is proposed for stabilization of rigid body dynamics (position, orientation and velocities) using unit dual quaternions, in which the dual quaternions and velocities are used for feedback. Specifically, both set-point stabilization and tracking control are addressed in this work. It is well-known that rigid body attitude control is subject to topological constraints which often result in discontinuous control to avoid the unwinding phenomenon. In contrast, the hybrid scheme allows the controlled system to be robust in the presence of uncertainties, which would otherwise cause chattering about the point of discontinuous control while also ensuring acceptable closed-loop response characteristics. The stability of the closed-loop system is guaranteed through a Lyapunov analysis and the use of invariance principles for hybrid systems. Simulation results for a rigid body model are presented to illustrate the performance of the proposed hybrid dual quaternion feedback control schemes.

I. Notation

The following notation and definitions are used throughout the paper:

- \mathbb{R}^n denotes n -dimensional Euclidean space.
- \mathbb{R} denotes the real numbers and $\mathbb{R}_{\geq 0}$ denotes the nonnegative real numbers; i.e., $\mathbb{R}_{\geq 0} = [0, \infty)$
- \mathbb{Z} denotes the integers. \mathbb{N} denotes the natural numbers including 0; i.e., $\mathbb{N} = \{0, 1, \dots\}$.
- \mathbb{B} denotes the closed unit ball, of appropriate dimension, in a Euclidean norm.
- Given a vector $x \in \mathbb{R}^n$, $|x|$ denotes the Euclidean vector norm.

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- Given a set S , \bar{S} denotes its closure. Given a point $x \in \mathbb{R}^n$, $|x|_S := \inf_{y \in S} |x - y|$.
- The equivalent notation $[x^\top \ y^\top]^\top$, and (x, y) is used for vectors.
- Given a vector $x \in \mathbb{R}^n$, $\nu(x) := [0 \ x^\top]^\top$.
- An $n \times n$ identity matrix is defined as I_n . An $n \times p$ zero vector/matrix is represented by $0_{n \times p}$.
- The unit quaternion with scalar part equal to one and the zero quaternion are given by $\mathbf{1} = (1, 0_{3 \times 1})$ and $\mathbf{0} = (0, 0_{3 \times 1})$, respectively.
- A function $\alpha : \mathbb{R}_{\geq 0} \rightarrow \mathbb{R}_{\geq 0}$ is said to belong to class- \mathcal{K} if it is continuous, zero at zero, and strictly increasing.
- A function $\alpha : \mathbb{R}_{\geq 0} \rightarrow \mathbb{R}_{\geq 0}$ is said to belong to class- \mathcal{K}_∞ if it belongs to class- \mathcal{K} and is unbounded.
- A function $\beta : \mathbb{R}_{\geq 0} \times \mathbb{R}_{\geq 0} \rightarrow \mathbb{R}_{\geq 0}$ is said to belong to class- \mathcal{KL} if it is nondecreasing in its first argument, nonincreasing in its second argument, and $\lim_{s \searrow 0} \beta(s, t) = \lim_{t \rightarrow \infty} \beta(s, t) = 0$.

II. Introduction

Rigid body control is often separated into two individual problems: attitude control (see [1–6]) and translational (point mass) control (see [7] and the references therein). However, for many practical applications that include robotics, computer graphics [8, 9], unmanned air vehicle control and spacecraft proximity operations [10–12] to name a few, these translational and rotational dynamics are often coupled. Hence, some recent research on controlling rigid body dynamics utilizes the Lie group $SE(3)$ for the configuration space (pose) of the rigid body and its tangent bundle $TSE(3)$ for the state space which includes velocities [13–15]. Tracking control of fully actuated vehicles is discussed in detail in [16]. Nevertheless, most of this work does not delve into the details of reconstructing the state of the system out of sensor measurements. In order to bypass this problem, a feedback law that directly utilizes vector measurements with the landmark-based control solution is presented in [17]. However, such strategies rely on continuous controllers while it has been shown in [18] that global asymptotic stabilization of a given set-point is not possible by means of continuous feedback. In order to solve this problem, continuous controllers based on the Morse-Lyapunov approach have been suggested in [2, 19] which result in almost global asymptotic stability, while discontinuous control laws have been proposed (see e.g. [20, 21]) to achieve global asymptotic stability. However, the latter are not robust to small measurement noise, as shown by [22]. Recent advances in hybrid control theory have shown that well-posed hybrid systems, namely, those satisfying the so-called hybrid basic conditions [23] are inherently robust to small measurement noise, making hybrid control techniques suitable candidates for the problem at hand. In fact, hybrid control strategies using both quaternion feedback and rotation matrix feedback have been proposed in [22, 24] and [25–29], respectively. Specifically on the tangent bundle $TSE(3)$ associated with the special Euclidean group $SE(3)$, [29] presents an application of hybrid control strategies to

under-actuated vehicles, while [30] designs hybrid control strategies for fully actuated rigid bodies with only landmark-based information.

It is a well known fact that global asymptotic stabilization of rigid body attitude is subject to topological constraints [31, 32]. Hence, a rigid body pose representation using unit dual quaternions (UDQs) inherits the same topological difficulties as the rigid body attitude parametrization using unit quaternions (see [33, 34] and the references therein). Specifically, a UDQ provides a dual cover for the elements in $SE(3)$, i.e., for every element in $SE(3)$, there are exactly two UDQs. Since such a representation of rigid body pose is non-unique, the control objective results in stabilizing a disconnected set of UDQ's representing the same rigid body position and orientation. Similar to the problem of rigid body attitude stabilization in $SO(3)$ [32], a continuous linear feedback law (as in [10, 34, 35]) results in the ‘unwinding’ phenomenon, while a discontinuous controller designed as in [36, 37] would not be robust to small measurement noise and nonlinear controllers may suffer in terms of performance. Hybrid feedback control [23] can overcome such topological obstructions and provide robust global solutions for the rigid body attitude stabilization problem [32]. In the case of full state measurements (i.e., position, orientation, linear and angular velocity measurements), [10] presents a continuous controller for rigid body pose stabilization. Results associated with the kinematic sub-problem of rigid body motion using hybrid hysteresis-based UDQs are presented in [33], while an improved version using a bimodal approach to reduce higher average settling time or energy consumption is presented in [38]. In addition, MPC-based dual quaternion spacecraft pose control is presented in [39, 40]. In this paper we adapt the hysteresis-based switching strategy of rigid body attitude presented in [31, 32] to the Unit Dual Quaternion (UDQ) parameterization of rigid body pose. Specifically, a complete solution for rigid body kinematic and kinetic control is presented using a hybrid hysteresis-based switching strategy. Considering that the full state, i.e, position, orientation, linear and angular velocity measurements are available for feedback, the following problems of interest are formalized in this paper:

- A general hybrid feedback control solution with dual quaternion and dual velocity feedback for a rigid body constant set-point pose stabilization is presented and its details are discussed in Section V.A. Unlike [33], where only the attitude and translational kinematics were treated, this paper treats hybrid control of both kinematics and kinetics.
- The problem of tracking a time-varying reference is discussed in detail in Section V.B. A hybrid control strategy to address a rigid body pose tracking a time-varying reference is formulated. As an improvement to results presented in [10, 36, 37], this paper establishes robust global asymptotic stability of rigid body set-point stabilization and time-varying reference tracking problems, respectively.
- Robustness of the proposed algorithms to uncertainties is discussed in Section V.C.
- Numerical examples for a rigid body set-point stabilization and time-varying reference tracking are

given in Section VI.

III. Preliminaries

A. Well-posed hybrid systems

Hybrid systems are dynamical systems with both continuous and discrete dynamics, where a hybrid system $\mathcal{H} = (C, f, D, g)$ is defined by the following objects:

- A mapping $f : \mathbb{R}^n \rightarrow \mathbb{R}^n$ called the *flow map*.
- A mapping $g : \mathbb{R}^n \rightarrow \mathbb{R}^n$ called the *jump map*.
- A set $C \subset \mathbb{R}^n$ called the *flow set*.
- A set $D \subset \mathbb{R}^n$ called the *jump set*.

The flow map f defines the continuous dynamics on the flow set C , while the jump map g defines the discrete dynamics on the jump set D . These objects are referred to as the data of the hybrid system \mathcal{H} . Given a state χ of the hybrid system \mathcal{H} , the notation χ^+ indicates the values of the state after the jump*. A solution ϕ to \mathcal{H} is given on extended time domain, called *hybrid time domain*, that is parametrized by the pairs (t, j) , where t is the ordinary time component and j is a discrete parameter that keeps track of the number of jumps; see [23]. Given a solution ϕ to \mathcal{H} , the notation $\text{dom } \phi$ represents its domain, which is a hybrid time domain. A solution to \mathcal{H} is said to be *nontrivial* if $\text{dom } \phi$ contains at least one point different from $(0, 0)$, *complete* if $\text{dom } \phi$ is unbounded, and *maximal* if it cannot be extended, i.e., it is not a truncated version of another solution. The set $\mathcal{S}_{\mathcal{H}}(\xi)$ denotes the set of all maximal solutions to \mathcal{H} from ξ .

B. Rigid body pose

The position and orientation of a rigid body with respect to a generic reference frame is defined by its relative position $p \in \mathbb{R}^3$ and its relative orientation $R \in SO(3)$ which represents a rotation from the body frame to the inertial frame. Namely, its position p and orientation R form an element (p, R) of the three-dimensional special Euclidean group $SE(3) := \mathbb{R}^3 \times SO(3)$. Given $(p, R) \in SE(3)$, a unit dual quaternion associated with it is given by [41]

$$\hat{q} = q_r + \epsilon q_t, \tag{1}$$

where

$$q_r = \begin{bmatrix} \eta_r \\ \mu_r \end{bmatrix} \in \mathcal{S}^3 : R = \mathcal{R}(q_r),$$

*Precisely, given a trajectory $(t, j) \mapsto \chi(t, j)$ to \mathcal{H} , $\chi^+ = \chi(t, j + 1)$ or $\chi^+ = g(\chi(t, j))$

and

$$q_t = \begin{bmatrix} \eta_t \\ \mu_t \end{bmatrix} = \frac{1}{2} \nu(p) \otimes q_r \in \mathbb{H},$$

where $\nu(p) = \begin{bmatrix} 0 & p^\top \end{bmatrix}^\top$, $p \in \mathbb{R}^3$ is the position of the center of mass in inertial frame. Notice that the position of the rigid body in body frame of reference is given by $\nu(p_b) = q_r^* \otimes \nu(p) \otimes q_r \in \mathbb{H}^v$, $p_b \in \mathbb{R}^3$. A list of basic UDQ operations are given in the Appendix.

IV. Problem description

Given an orthonormal inertial frame $\{I\}$ and an orthonormal body frame $\{B\}$, fixed to the rigid body, its dynamic equations of motion in dual-quaternion representation [10, 38] are given by (see Appendix for details)

$$\begin{aligned} \dot{\hat{q}}_b &= \frac{1}{2} \hat{q}_b \otimes \nu(\hat{\omega}_b) \\ M \star \nu(\hat{\omega}_b^s) &= \hat{u} - \nu(\hat{\omega}_b) \times (M \star \nu(\hat{\omega}_b^s)) \end{aligned} \quad (2)$$

where $\nu(\hat{\omega}_b) = (0 + \epsilon 0, \hat{\omega}_b) \in \widehat{\mathbb{H}}^v$, $\nu(\hat{\omega}_b^s) = (0 + \epsilon 0, \hat{\omega}_b^s) \in \widehat{\mathbb{H}}^v$, $\hat{\omega}_b^s = v_b + \epsilon \omega_b$, $\hat{\omega}_b = \omega_b + \epsilon v_b$, $\omega_b, v_b \in \mathbb{R}^3$ are the linear and angular velocities of the rigid body with respect to the inertial frame $\{I\}$ represented in the body frame $\{B\}$, respectively; and $\hat{u} = \nu(F) + \epsilon \nu(\tau) \in \widehat{\mathbb{H}}^v$, where $F \in \mathbb{R}^3$ represents control forces and $\tau \in \mathbb{R}^3$ represents control torques applied to the rigid body in its frame of reference. The dual inertia matrix (2) is given by

$$M = \begin{bmatrix} 1 & 0_{1 \times 3} & 0 & 0_{1 \times 3} \\ 0_{3 \times 1} & mI_3 & 0_{3 \times 1} & 0_{3 \times 3} \\ 0 & 0_{1 \times 3} & 1 & 0_{1 \times 3} \\ 0_{3 \times 1} & 0_{3 \times 3} & 0_{3 \times 1} & J \end{bmatrix}, \quad (3)$$

where $m \in \mathbb{R}$ is the mass of the rigid body, $J = J^\top > 0$, $J \in \mathbb{R}^{3 \times 3}$ is the mass moment of inertia of the body about its center of mass written in the body frame. Since the mass m is positive and the inertia matrix J is a real symmetric positive definite matrix, the dual inertia matrix M in the above formulation is invertible.

With this dynamic model, the main goal in this paper is to design a controller that asymptotically stabilizes the rigid body pose to a desired constant set-point given by $(\hat{q}_d, \hat{\mathbf{0}}) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$ or time-varying reference position, orientation, and velocities, $t \mapsto (\hat{q}_d(t), \nu(\hat{\omega}_d(t))) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$, where $\hat{\omega}_d(t)$ is the dual velocity of the desired frame $\{D\}$ with respect to the inertial frame $\{I\}$ represented in the body frame $\{B\}$.

To formally present the problem, let us define the dual quaternion and dual velocity error variables of the

body frame $\{B\}$ with respect to the desired frame $\{D\}$ resolved into $\{B\}$ as

$$\bar{q} := \hat{q}_d^* \otimes \hat{q}_b \in \widehat{\mathcal{S}}^3, \quad \nu(\bar{\omega}) := \nu(\hat{\omega}_b) - \nu(\hat{\omega}_d) \in \widehat{\mathbb{H}}^v. \quad (4)$$

Differentiating the above error variables and following [34, 42], yields the error dynamics

$$\begin{aligned} \dot{\bar{q}} &= \frac{1}{2} \bar{q} \otimes \nu(\bar{\omega}) \\ \nu(\dot{\bar{\omega}}^s) &= M^{-1} \star (\hat{u} - \nu(\hat{\omega}_b) \times (M \star \nu(\hat{\omega}_b^s))) - \nu(\dot{\hat{\omega}}_d^s) \end{aligned} \quad (5)$$

Then, in these error coordinates, convergence to the desired constant set-point $(\hat{q}_d, \hat{\mathbf{0}})$ or to the time-varying reference $t \mapsto (\hat{q}_d(t), \nu(\hat{\omega}_d(t))) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$ reduces to \bar{q} converging to the unit dual quaternion $\pm \hat{\mathbf{1}}$ and $\nu(\bar{\omega})$ converging to the dual quaternion $\hat{\mathbf{0}}$. With this reformulation, the problem we solve in this paper is stated as follows.

For scenarios with full state feedback, i.e., the entire state $(\hat{q}_b, \hat{\omega}_b)$ is available for feedback,

Problem 1. Given a constant set-point reference pose $(\hat{q}_d, \hat{\mathbf{0}}) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$; or

Problem 2. Given a reference pose trajectory $t \mapsto (\hat{q}_d(t), \nu(\hat{\omega}_d(t))) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$;

design a control law assigning \hat{u} in (2) such that the resulting closed-loop system satisfies the following properties:

- 1) **Stability:** trajectories to the closed-loop system in error coordinates $(\bar{q}, \nu(\bar{\omega}))$ are such that \bar{q} stays close to either $\hat{\mathbf{1}}$ or $-\hat{\mathbf{1}}$ and $\nu(\bar{\omega})$ stays close to zero when they start close to each respective point;
- 2) **Attractivity:** In the error coordinates $(\bar{q}, \nu(\bar{\omega}))$, the \bar{q} component converges to $\hat{\mathbf{1}}$ or $-\hat{\mathbf{1}}$, with zero linear and angular velocity $\nu(\bar{\omega})$.
- 3) **Robustness:** For each compact set of initial conditions and level of closeness to reference set-point, there exists nonzero perturbation to the closed-loop system such that for each initial position, orientation, linear and angular velocities of the rigid body in the said compact set, the resulting trajectories converge to nearby the set-point, with desired level of closeness.

V. Hybrid Feedback Control and Stability

Given the rigid body kinematics and dynamics in error coordinates in (5), due to a desired constant or a time-varying structure of the reference given by $(\hat{q}_d, \hat{\mathbf{0}}) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$, $t \mapsto (\hat{q}_d(t), \nu(\hat{\omega}_d(t))) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$, respectively, in this section, we present the hybrid feedback control design for each of these cases separately.

A. Problem 1: Rigid body constant set-point pose stabilization

With the rigid body kinematics and dynamics in error coordinates in (5), as in the scenario of **Problem 1**, consider that a constant set-point reference pose $(\hat{q}_d, \hat{\mathbf{0}}) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$ is given and the output of rigid body

dynamics (defined in (2)) $y = (\hat{q}_b, \hat{\omega}_b)$ is available for feedback. Hence, the error vector $(\bar{q}, \bar{\omega})$ defined in (4) is available for feedback. In addition, for the set-point stabilization problem since the desired dual velocity $\nu(\hat{\omega}_d) = \hat{\mathbf{0}}$, the rigid body kinematics and dynamics in error coordinates in (5) can be rewritten as follows:

$$\begin{aligned}\dot{\bar{q}} &= \frac{1}{2}\bar{q} \otimes \nu(\bar{\omega}), \\ \nu(\bar{\omega}^s) &= M^{-1} \star (\hat{u} - \nu(\bar{\omega}) \times (M \star \nu(\bar{\omega}^s))).\end{aligned}\tag{6}$$

A dual quaternion-based control law for such a system in (6) is presented in [34, Theorem 1, equation (13)] which, suffers from topological obstructions. To overcome this limitation and solve **Problem 1**, inspired by the formulation presented in [32], a dynamic feedback that depends on the logic variable $h \in \{-1, 1\} =: \mathcal{Q}$, is proposed. The proposed hybrid controller is given as follows:

$$\begin{aligned}\dot{h} &= 0 & (\bar{q}, \nu(\bar{\omega}), h) \in C, \\ h^+ &= -h & (\bar{q}, \nu(\bar{\omega}), h) \in D, \\ \hat{u} &= I_u \kappa(\bar{q}, \nu(\bar{\omega}), h)\end{aligned}\tag{7}$$

where $I_u := \begin{bmatrix} 0 & 0_{1 \times 3} \\ 0_{3 \times 1} & I_3 \end{bmatrix}$,

$$\kappa(\bar{q}, \nu(\bar{\omega}), h) := -hk_p(\bar{q}^* \otimes (h\bar{q}^s - \hat{\mathbf{1}}^s)) - k_d\nu(\bar{\omega}^s),\tag{8}$$

$k_p, k_d > 0$,

$$\begin{aligned}C &= \{(\bar{q}, \nu(\bar{\omega}), h) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v \times \mathcal{Q} : h\eta_r \geq -\delta\}, \\ D &= \{(\bar{q}, \nu(\bar{\omega}), h) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v \times \mathcal{Q} : h\eta_r \leq -\delta\},\end{aligned}\tag{9}$$

with $\delta \in (0, 1)$ and η_r is the scalar part of rotational error quaternion $q_r \in \mathcal{S}^3$, where $\bar{q} = q_r + \epsilon q_t$. Hence, the hybrid closed-loop model of the rigid body error kinematics and dynamics includes system (6) and the hybrid feedback controller (7)-(9). The closed-loop system denoted by $\mathcal{H} = (C, f, D, g)$ has state[†] $\xi = (\bar{q}, \nu(\bar{\omega}), h) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v \times \mathcal{Q} =: \mathcal{X}$ and hybrid dynamics

$$\begin{aligned}\dot{\xi} &= f(\xi) & \xi \in C, \\ \xi^+ &= g(\xi) & \xi \in D.\end{aligned}\tag{10}$$

[†]As in previous work using models in terms of unit dual quaternions [10, 34, 35] and closed-loop systems with states using unit quaternions and logic variables [24, 32], we treat the state space of the closed-loop system, namely, \mathcal{X} , as a set embedded in a large enough Euclidean space. As in those references, this embedding allow us to employ notions for closedness of sets and continuity of maps that are standard in Euclidean spaces.

Details on hybrid system modeling are presented in Section III.A. The flow and jump sets satisfy $C \cup D = \mathcal{X}$ and the maps $f : \mathcal{X} \rightarrow \mathcal{X}$ and $g : \mathcal{X} \rightarrow \mathcal{X}$ are given by

$$f(\xi) := \begin{bmatrix} \frac{1}{2}\bar{q} \otimes \nu(\bar{\omega}) \\ M^{-1} \star (I_u \kappa(\xi) - \nu(\bar{\omega}) \times (M \star \nu(\bar{\omega}^s))) \\ 0 \end{bmatrix}, \quad g(\xi) := \begin{bmatrix} \bar{q} \\ \nu(\bar{\omega}^s) \\ -h \end{bmatrix}. \quad (11)$$

Due to the design of the hybrid feedback (7)-(9), this hybrid system renders the compact set

$$\mathcal{A} = \{\xi \in \mathcal{X} : \bar{q} = h\hat{\mathbf{1}}, \nu(\bar{\omega}^s) = \hat{\mathbf{0}}\}, \quad (12)$$

globally asymptotically stable (details of this result are given in Theorem V.2 below). Note that for a constant set-point stabilization problem, the linear and angular velocity of the fixed frame $\nu(v_d) = \mathbf{0}$, $\nu(\omega_d) = \mathbf{0}$. Hence, in other words, the set \mathcal{A} represents the desired rigid body pose error $\bar{q} = q_r + \epsilon q_t = h\hat{\mathbf{1}}$, and dual velocity error $\nu(\bar{\omega}^s) = \nu(v_b - v_d) + \epsilon \nu(\omega_b - \omega_d) = \hat{\mathbf{0}}$, i.e., the desired pose $q_r = \mathbf{1}$, $q_t = \mathbf{0}$, angular velocity $\nu(\omega_b) = \mathbf{0}$, and linear velocity $\nu(v_b) = \mathbf{0}$.

Remark V.1 Given the desired position, orientation, and velocities $(\hat{q}_d, \nu(\hat{\omega}_d)) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$, the first term in the equation (8) can be written as

$$-hk_p(\bar{q}^* \otimes (h\bar{q}^s - \hat{\mathbf{1}}^s)) = -k_p(h\bar{q}^* \otimes (hq_t + \epsilon(hq_r - \mathbf{1}))). \quad (13)$$

Using the quaternion multiplication rule, equation (13) can be rewritten as follows.

$$-k_p h(\bar{q}^* \otimes (h\bar{q}^s - \hat{\mathbf{1}}^s)) = -k_p \begin{bmatrix} \eta_r \eta_t + \mu_r^\top \mu_t \\ \eta_r \mu_t - \eta_t \mu_r - \mu_r \times \mu_t \end{bmatrix} + \epsilon k_p \begin{bmatrix} 1 - h\eta_r + \eta_t^2 + \mu_t^\top \mu_t \\ -h\mu_r \end{bmatrix}. \quad (14)$$

Therefore, the output of the dynamic feedback (7)-(9), using (4), can be re-written as following.

$$\hat{u} = \begin{bmatrix} 0 \\ -k_p(\eta_r \mu_t - \eta_t \mu_r) + k_p(\mu_r \times \mu_t) - k_d(v_b - v_d) \end{bmatrix} + \epsilon \begin{bmatrix} 0 \\ -hk_p \mu_r - k_d(\omega_b - \omega_d) \end{bmatrix}. \quad (15)$$

Therefore, equating the input $\hat{u} = \nu(F) + \nu(\tau) \in \widehat{\mathbb{H}}^v$ to (15) results in the following expression for the force $F \in \mathbb{R}^3$ and torque $\tau \in \mathbb{R}^3$:

$$\begin{aligned} F &= -k_p(\eta_r \mu_t - \eta_t \mu_r) + k_p(\mu_r \times \mu_t) - k_d(v_b - v_d), \\ \tau &= -hk_p \mu_r - k_d(\omega_b - \omega_d). \end{aligned} \quad (16)$$

Note that for the set-point stabilization problem, namely, **Problem 1**, the linear and angular velocity v_d and ω_d of the fixed desired frame satisfy $v_d = \mathbf{0}_{3 \times 1}$, $\omega_d = \mathbf{0}_{3 \times 1}$ in (16).

Next, the hybrid closed-loop system \mathcal{H} satisfies the hybrid basic conditions (see [23, Proposition 6.10]) and our main result is as follows.

Theorem V.2 *The set \mathcal{A} in (12) is globally asymptotically stable for the closed-loop system \mathcal{H} .*

Proof: For the hybrid closed-loop system (10), we first show that every complete solution to it converges to \mathcal{A} . For this purpose, we use the invariance principle for hybrid systems in [23] for which \mathcal{H} has to satisfy the hybrid basic conditions, which is already the case from the hybrid system \mathcal{H} formulation. After that, since \mathcal{H} satisfies the hybrid basic conditions, following [23, Proposition 6.10], we can conclude every maximal solution to the hybrid system is complete, in this way showing the asymptotic stability of \mathcal{A} .

Now to show convergence of complete solutions to \mathcal{A} , consider the Lyapunov function candidate $V : \mathcal{X} \rightarrow \mathbb{R}$ given by

$$V(\xi) = \mathbf{1}^\top \bar{V}(\xi) \quad \forall \xi \in \mathcal{X} \quad (17)$$

where $\bar{V} : \mathcal{X} \rightarrow \mathbb{H}^s$ is defined as

$$\bar{V}(\xi) := k_p(h\bar{q} - \hat{\mathbf{1}}) \circ (h\bar{q} - \hat{\mathbf{1}}) + \frac{1}{2}\nu(\bar{\omega}^s) \circ (M \star \nu(\bar{\omega}^s)) \quad \forall \xi \in \mathcal{X},$$

$\hat{\mathbf{1}} = \mathbf{1} + \epsilon\mathbf{0}$, $\mathbf{1} = (1, 0_{3 \times 1})$, and ‘ \circ ’ operator for the UDQs is defined in item 8)d) in Appendix. With $q_r = (\eta_r, \mu_r) \in \mathcal{S}^3$, $q_t = (0, \mu_t) \in \mathbb{H}^v$ as defined in (1), $\nu(\omega) = (0, \omega_b) \in \mathbb{H}^v$, $\nu(v) = (0, v_b) \in \mathbb{H}^v$, as defined in (4), since $q_r \in \mathcal{S}^3$, $\eta_r^2 + \mu_r^\top \mu_r = 1$ and with $h^2 = 1$, (17) can be simplified as

$$V(\xi) = 2k_p(1 - h\eta_r) + k_p(\mu_t^\top \mu_t) + \frac{1}{2}(mv_b^\top v_b + \omega_b^\top J\omega_b). \quad (18)$$

The Lyapunov function in (18) satisfies $V(\xi) = 0$ for all $\xi \in \mathcal{A}$, $V(\xi) > 0$ for all $\xi \notin \mathcal{A}$. In addition, for any $c > 0$, there exists a $r > 0$ such that $V(\xi) > c$ whenever $|\xi| > r$. Thus the set $\Omega_c := \{\xi \in \mathcal{X} : V(\xi) \leq c\}$ is compact for every $c > 0$.

Next, the time derivative of the Lyapunov function candidate V in (17) along the flows is given by [‡]

$$\begin{aligned} \frac{d}{dt}V(\xi) &= \mathbf{1}^\top \left(\frac{d}{dt}\bar{V}(\xi) \right) \\ &= \mathbf{1}^\top \left(k_p h \frac{d}{dt}(h\bar{q} - \hat{\mathbf{1}}) \circ \frac{d}{dt}(\bar{q}) + k_p h \frac{d}{dt}(\bar{q}) \circ (h\bar{q} - \hat{\mathbf{1}}) \right. \\ &\quad \left. + \frac{1}{2}\nu(\bar{\omega}^s) \circ (M \star \frac{d}{dt}(\nu(\bar{\omega}^s))) + \frac{1}{2}\frac{d}{dt}(\nu(\bar{\omega}^s)) \circ (M \star \nu(\bar{\omega}^s)) \right). \end{aligned} \quad (19)$$

Next, using the properties in items 10)c), 10)d) of Appendix, respectively,

$$\frac{d}{dt}V(\xi) = \mathbf{1}^\top (2k_p h \frac{d}{dt}(h\bar{q} - \hat{\mathbf{1}}) \circ \frac{d}{dt}(\bar{q}) + \nu(\bar{\omega}^s) \circ (M \star \frac{d}{dt}(\nu(\bar{\omega}^s)))). \quad (20)$$

[‡] By $\frac{d}{dt}V(\xi)$ we mean the inner product between the gradient of V and the vector field f governing the continuous change of ξ given in (10).

With f in (10), and property 13) in Appendix,

$$\begin{aligned} \frac{d}{dt}V(\xi) &= \mathbf{1}^\top(2k_ph(h\bar{q} - \hat{\mathbf{1}}) \circ (\frac{1}{2}\bar{q} \otimes \nu(\bar{\omega})) \\ &\quad + \nu(\bar{\omega}^s) \circ M \star M^{-1}(I_u\kappa(\xi) - \nu(\hat{\omega}_b) \times (M \star \nu(\hat{\omega}_b^s))))), \end{aligned} \quad (21)$$

for each $\xi \in C$. Given $\bar{q}_1, \bar{q}_2, \bar{q}_3 \in \widehat{\mathbb{H}}$, respectively, from Appendix, following the property in item 10)a), the first term in (21) can be written as follows:

$$k_ph(h\bar{q} - \hat{\mathbf{1}}) \circ (\bar{q} \otimes \nu(\bar{\omega})) = \nu(\bar{\omega}^s) \circ (\bar{q}^* \otimes k_ph(h\bar{q}^s - \epsilon\mathbf{1})). \quad (22)$$

Next, the second term in (21) with $\hat{u} = I_u\kappa(\xi)$ is given by

$$\nu(\bar{\omega}^s) \circ (\hat{u} - \nu(\bar{\omega}) \times (M \star \nu(\bar{\omega}^s))) = \nu(\bar{\omega}^s) \circ \hat{u} - \nu(\bar{\omega}^s) \circ (\nu(\bar{\omega}) \times (M \star \nu(\bar{\omega}^s))). \quad (23)$$

Using the operation in items 10)b) and 10)f) along with the cross product operation of the dual quaternion in item 10)g) of Appendix, the second term in (23) results in the following:

$$\nu(\bar{\omega}^s) \circ (\nu(\bar{\omega}) \times (M \star \nu(\bar{\omega}^s))) = \hat{\mathbf{0}}.$$

Then, combining these steps, we have,

$$\begin{aligned} \nu(\bar{\omega}^s) \circ (\hat{u} - \nu(\bar{\omega}) \times (M \star \nu(\bar{\omega}^s))) &= \nu(\bar{\omega}^s) \circ I_u\kappa(\xi) - \nu(\bar{\omega}^s) \circ (\nu(\bar{\omega}) \times (M \star \nu(\bar{\omega}^s))), \\ &= \nu(\bar{\omega}^s) \circ I_u\kappa(\xi). \end{aligned} \quad (24)$$

Therefore, from (22) and (23), since $k_p > 0$ is a constant and $h \in \mathcal{Q}$,

$$\begin{aligned} \frac{d}{dt}V(\xi) &= \mathbf{1}^\top(\nu(\bar{\omega}^s) \circ (\bar{q}^* \otimes k_ph(h\bar{q}^s - \epsilon\mathbf{1})) + \nu(\bar{\omega}^s) \circ I_u\kappa(\xi)), \\ &= \mathbf{1}^\top(\nu(\bar{\omega}^s) \circ (k_ph\bar{q}^* \otimes (h\bar{q}^s - \epsilon\mathbf{1}) + I_u\kappa(\xi))). \end{aligned}$$

From (8), since $I_u\kappa(\xi) = I_u(-k_ph\bar{q}^* \otimes (h\bar{q}^s - \hat{\mathbf{1}}^s) - k_d\nu(\bar{\omega}^s)) \in \widehat{\mathbb{H}}$, we get

$$\begin{aligned} \frac{d}{dt}V(\xi) &= \mathbf{1}^\top(\nu(\bar{\omega}^s) \circ (k_ph\bar{q}^* \otimes (h\bar{q}^s - \epsilon\mathbf{1}) + I_u(-k_ph\bar{q}^* \otimes (h\bar{q}^s - \epsilon\mathbf{1})))) \\ &\quad - \mathbf{1}^\top(k_d\nu(\bar{\omega}^s) \circ \nu(\bar{\omega}^s)), \end{aligned} \quad (25)$$

where $I_u = \begin{bmatrix} 0 & 0_{1 \times 3} \\ 0_{3 \times 1} & I_3 \end{bmatrix}$, $k_p, k_d > 0$.

With the ‘ \circ ’ operator defined in item 8)d) in Appendix, the non-velocity term in (25), using the definitions

$v = v_b - v_d$, $\omega = \omega_b - \omega_d$ (for notational simplicity) reduces to

$$\frac{d}{dt}V(\xi) = -\mathbf{1}^\top(k_d\nu(\bar{\omega}^s) \circ \nu(\bar{\omega}^s)) = -k_d\omega^\top\omega - k_dv^\top v. \quad (26)$$

Therefore, from (26), defining, for each $\xi \in C$,

$$u_C(\xi) := \begin{cases} -k_d \omega^\top \omega - k_d v^\top v & \text{if } \xi \in C \\ -\infty & \text{otherwise,} \end{cases} \quad (27)$$

we can see that $\langle \nabla V(\xi), f(\xi) \rangle = u_C(\xi) \leq 0$.

Next, at jumps, for each $\xi \in D$, the Lyapunov function candidate V in (17) changes as follows:

$$V(g(\xi)) - V(\xi) = \mathbf{1}^\top (k_p((-h\bar{q} - \hat{\mathbf{1}}) \circ (-h\bar{q} - \hat{\mathbf{1}})) - ((h\bar{q} - \hat{\mathbf{1}}) \circ (h\bar{q} - \hat{\mathbf{1}})))$$

Given $\bar{q} := q_r + \epsilon q_t$, where $q_r = (\eta_r, \mu_r) \in \mathcal{S}^3$, $q_t = (0, \mu_t) \in \mathbb{H}^v$ are defined in (1),

$$\begin{aligned} V(g(\xi)) - V(\xi) &= \mathbf{1}^\top (k_p((-h(q_r + \epsilon q_t) - \hat{\mathbf{1}}) \circ (-h(q_r + \epsilon q_t) - \hat{\mathbf{1}})) - ((h(q_r + \epsilon q_t) - \hat{\mathbf{1}}) \circ (h(q_r + \epsilon q_t) - \hat{\mathbf{1}}))) \\ &= k_p(\mu_r \cdot \mu_r + (h\eta_r + 1)^2 - \mu_r \cdot \mu_r - (h\eta_r - 1)^2) = 4k_p h\eta_r. \end{aligned} \quad (28)$$

Since, for each point ξ in D , $h\eta_r \leq -\delta$,

$$V(g(\xi)) - V(\xi) \leq -4k_p \delta.$$

Defining, for each $\xi \in D$,

$$u_D(\xi) := \begin{cases} -4k_p \delta & \text{if } \xi \in D \\ -\infty & \text{otherwise,} \end{cases} \quad (29)$$

we have $V(g(\xi)) - V(\xi) = u_D(\xi) < 0$ for all $\xi \in D \setminus \mathcal{A}$.

Completeness of maximal solutions: We have that $u_C(\xi)$ and $u_D(\xi)$ are nonpositive for all $\xi \in \mathcal{X}$. And hence every solution $\phi \in \mathcal{S}_{\mathcal{H}}(\phi(0, 0))$, where $\phi(0, 0) \in \mathcal{X}$ to the hybrid system in (10) remains in \mathcal{X} for all $(t, j) \in \text{dom}(\phi)$. Also, \mathcal{A} is compact, and the Lyapunov function V is positive definite relative to \mathcal{A} , the sublevel set $\Omega_c := \{\xi \in \mathcal{X} : V(\xi) \leq c\}$ is compact for every $c > 0$ and V is non-increasing along the solutions of \mathcal{H} . These results show that any solution ϕ to the hybrid system \mathcal{H} is bounded and do not blow up in finite time. Also, $g(D) \subset C \cup D$ which shows that the every solution ϕ to system \mathcal{H} does not jump out of $C \cup D$. Therefore, from [23, Proposition 2.10], since conditions (b) and (c) therein are not satisfied, we conclude that every maximal solution to the closed-loop system \mathcal{H} is complete.

Invariance principle for hybrid systems: The growth of V along the solutions to \mathcal{H} is bounded by $u_C(\xi)$ and $u_D(\xi)$ on \mathcal{X} . Since \mathcal{H} satisfies the hybrid basic conditions and V in (17) is continuous, the invariance principle for hybrid systems in [23, Theorem 8.2] implies that every precompact (complete and bounded)

solution to the hybrid system (10) converges to the largest weakly invariant set W contained in

$$V^{-1}(a) \cap \mathcal{X} \cap [\overline{u_C^{-1}(0)} \cup (u_D^{-1}(0) \cap g(u_D^{-1}(0)))] \quad (30)$$

for some $a \in \mathbb{R}_{\geq 0}$. Note that for every point in $\widehat{\mathcal{S}}^3$, $\bar{\mu} = \mu_r + \epsilon\mu_t = 0_{3 \times 1} + \epsilon 0_{3 \times 1}$ implies $\bar{\eta} = \eta_r + \epsilon\eta_t = \pm 1 + \epsilon 0$. By evaluating the dynamics (10) along solutions that remain in (30), we have that $\nu(\bar{\omega}) \equiv \hat{\mathbf{0}}$. Therefore, with f in (10) and the expression of input \hat{u} in (15), $\nu(\bar{\omega}^s) \equiv \hat{\mathbf{0}}$ implies $\bar{\mu} = \mu_r + \epsilon\mu_t = 0_{3 \times 1} + \epsilon 0_{3 \times 1}$ and since $h\eta_r \geq -\delta$ with $\delta \in (1, 0)$, then for all $\xi \in \mathcal{X} \cap \overline{u_C^{-1}(0)}$, $\bar{q} = h\hat{\mathbf{1}}$. Hence

$$\begin{aligned} W &\subset \{\xi \in \mathcal{X} : h\eta_r \geq -\delta, \eta_r = \pm 1, \mu_r = 0_{3 \times 1}, \eta_t = 0, \mu_t = 0_{3 \times 1}, \nu(\bar{\omega}^s) = \hat{\mathbf{0}}\} \cap V^{-1}(a) \\ &\subset \{\xi \in \mathcal{X} : \bar{q} = h\hat{\mathbf{1}}, \nu(\bar{\omega}^s) = \hat{\mathbf{0}}\} \cap V^{-1}(a) \end{aligned}$$

Then, since the only invariant set is for $a = 0$, (30) with $a = 0$ is contained in \mathcal{A} , i.e.,

$$W \subset \{\xi \in \mathcal{X} : \bar{q} = h\hat{\mathbf{1}}, \nu(\bar{\omega}^s) = \hat{\mathbf{0}}\} \cap V^{-1}(0) \subset \mathcal{A}.$$

Since every maximal solution to \mathcal{H} is precompact, then each maximal solution ϕ to \mathcal{H} converges to \mathcal{A} . We conclude that \mathcal{A} is globally attractive for the hybrid system \mathcal{H} . Since the function V in (17) is positive-definite relative to \mathcal{A} and nonincreasing along the solutions of \mathcal{H} , then \mathcal{A} is stable for the closed-loop hybrid system. Hence, we conclude that the set \mathcal{A} is globally asymptotically stable for the hybrid system \mathcal{H} . \blacksquare

B. Problem 2: Rigid body time-varying reference pose tracking

Let us consider the rigid body dynamics between an orthonormal inertial frame $\{I\}$ and an orthonormal body frame $\{B\}$ as outlined in Section IV. Let $t \mapsto x_d(t) := (p_d, q_d, v_d, \omega_d)(t)$ denote a smooth trajectory evolving on $\mathcal{S}^3 \times \mathbb{R}^9$ for all $t \geq 0$ satisfying the following assumption.

Assumption V.3 *Let $\pi : \mathcal{S}^3 \times \mathbb{R}^9 \rightarrow \mathcal{S}^3 \times \mathbb{R}^3$ denote the canonical projection of $\mathcal{S}^3 \times \mathbb{R}^9$ on to $\mathcal{S}^3 \times \mathbb{R}^3$. The reference trajectory $t \mapsto x_d(t) := (p_d, q_d, v_d, \omega_d)(t)$ is a complete and bounded solution to $\dot{x}_d = \zeta(x_d)$ satisfying*

$$\frac{d}{dt} \pi(p_d(t), q_d(t), v_d(t), \omega_d(t)) = (v_d(t) - S(\omega_d(t))p_d(t), \frac{1}{2}q_d(t) \otimes \omega_d(t)) \quad (31)$$

for each $t \geq 0$ and for some continuously differentiable vector field ζ on $\mathcal{S}^3 \times \mathbb{R}^9$.

To this trajectory $t \mapsto x_d(t)$ satisfying Assumption V.3, for each $t \geq 0$ one may associate a desired reference frame $\{D\}$. The origin of such a desired reference frame is located at $p_d(0) \in \mathbb{R}^3$ with orientation given by $q_d(0) \in \mathcal{S}^3$. In addition, a unit dual quaternion associated with this desired reference frame is given by [41]

$$t \mapsto \hat{q}_d(t) := q_d(t) + \epsilon q_{d_i}(t), \quad (32)$$

where $t \mapsto q_{d_i}(t) = \frac{1}{2}q_d(t) \otimes \nu(p_d(t)) \in \mathbb{H}^v$ for all $t \geq 0$. With this desired frame reference trajectory, the

main goal in this section is as follows:

Problem 2: Given a reference trajectory $t \mapsto (\hat{q}_d(t), \hat{\omega}_d(t)) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$, design a control law as a function of the sensor outputs and the reference trajectory $t \mapsto (\hat{q}_d(t), \hat{\omega}_d(t)) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$ such that

$$\lim_{t \rightarrow \infty} \bar{q}(t) = \pm \hat{\mathbf{1}}, \quad \lim_{t \rightarrow \infty} \bar{\omega}(t) = \hat{\mathbf{0}}$$

for all initial conditions, where

$$\bar{q} = \hat{q}_d^* \otimes \hat{q}_b \in \widehat{\mathcal{S}}^3, \quad \nu(\bar{\omega}) := \nu(\hat{\omega}_b) - \nu(\hat{\omega}_d) \in \widehat{\mathbb{H}}^v, \quad (33)$$

$(\hat{q}_b, \hat{\omega}_b) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v$ is the state of the orthonormal body frame $\{B\}$ outlined in Section IV. Differentiating the error variables in (33), following [34, 42], the dynamics of the error variables are given as follows:

$$\begin{aligned} \dot{\bar{q}} &= \frac{1}{2} \bar{q} \otimes \nu(\bar{\omega}) \\ \nu(\dot{\bar{\omega}}^s) &= M^{-1} \star (\hat{u} - (\nu(\bar{\omega}) + \nu(\hat{\omega}_d)) \times (M \star (\nu(\bar{\omega}^s) + \nu(\hat{\omega}_d^s))) - M \star \nu(\hat{\omega}_d^s)), \end{aligned} \quad (34)$$

where $\hat{u} \in \widehat{\mathbb{H}}^v$ is the total dual force resolved into the body frame. Therefore, to solve **Problem 2**, let us consider a hybrid feedback, similar to the hybrid controller in (7), that depends on the logic variable $h \in \{-1, 1\} =: \mathcal{Q}$, along with a feedforward term that depends on a reference input $\mathbb{S}(\nu(\hat{\omega}_d^s), \nu(\hat{\omega}_d^s)) \in \widehat{\mathbb{H}}^v \times \widehat{\mathbb{H}}^v$, given as follows:

$$\begin{aligned} \dot{h} &= 0 & (\bar{q}, \nu(\bar{\omega}), h) \in C, \\ h^+ &= -h & (\bar{q}, \nu(\bar{\omega}), h) \in D, \\ \hat{u} &= I_u \tilde{\kappa}(\bar{q}, \nu(\bar{\omega}), h, \nu(\hat{\omega}_d^s), \nu(\hat{\omega}_d^s)) \end{aligned} \quad (35)$$

where $I_u = \begin{bmatrix} 0 & 0_{1 \times 3} \\ 0_{3 \times 1} & I_3 \end{bmatrix}$,

$$\tilde{\kappa}(\bar{q}, \nu(\bar{\omega}), h, \nu(\hat{\omega}_d^s), \nu(\hat{\omega}_d^s)) := \kappa_{fb}(\bar{q}, \nu(\bar{\omega}), h, \nu(\hat{\omega}_d^s)) + \kappa_{ff}(\nu(\hat{\omega}_d^s), \nu(\hat{\omega}_d^s)), \quad (36)$$

the terms

$$\begin{aligned} \kappa_{fb}(\bar{q}, \nu(\bar{\omega}), h, \nu(\hat{\omega}_d^s)) &= \kappa(\bar{q}, \nu(\bar{\omega}), h) + \nu(\bar{\omega}) \times (M \star \nu(\hat{\omega}_d^s)) + \nu(\hat{\omega}_d) \times (M \star \nu(\bar{\omega}^s)), \\ \kappa_{ff}(\nu(\hat{\omega}_d^s), \nu(\hat{\omega}_d^s)) &= \nu(\hat{\omega}_d) \times (M \star \nu(\hat{\omega}_d^s)) + M \star \nu(\hat{\omega}_d^s), \end{aligned}$$

$k_p, k_d > 0$, $\kappa(\bar{q}, \nu(\bar{\omega}), h)$ is given in (8),

$$\begin{aligned} C &= \{(\bar{q}, \nu(\bar{\omega}), h) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v \times \mathcal{Q} : h\eta_r \geq -\delta\}, \\ D &= \{(\bar{q}, \nu(\bar{\omega}), h) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v \times \mathcal{Q} : h\eta_r \leq -\delta\}, \end{aligned} \quad (37)$$

[§]we consider that the reference input is generated on the hybrid time domain $(t, j) \mapsto (\nu(\hat{\omega}_d^s(t, j)), \nu(\hat{\omega}_d^s(t, j)))$

with $\delta \in (0, 1)$ and η_r is the scalar part of rotational error quaternion $q_r \in \mathcal{S}^3$.

The hybrid closed-loop model for the rigid body tracking error kinematics and dynamics includes system (34) and the hybrid controller given in (35)-(37). The closed-loop system denoted by $\mathcal{H}_T = (C, f, D, g)$ has state $\xi = (\bar{q}, \nu(\bar{\omega}), h) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v \times \mathcal{Q} =: \mathcal{X}$ and hybrid dynamics represented by (10). The flow and jump sets satisfy $C \cup D = \mathcal{X}$ and due to the design of the controller (35)-(37), the maps $f : \mathcal{X} \rightarrow \mathcal{X}$ and $g : \mathcal{X} \rightarrow \mathcal{X}$ are given by

$$f(\xi) := \begin{bmatrix} \frac{1}{2}\bar{q} \otimes \nu(\bar{\omega}) \\ M^{-1} \star (I_u \tilde{\kappa}(\xi) - \nu(\bar{\omega}) \times (M \star \nu(\bar{\omega}^s))) \\ 0 \end{bmatrix}, \quad g(\xi) := \begin{bmatrix} \bar{q} \\ \nu(\bar{\omega}^s) \\ -h \end{bmatrix}. \quad (38)$$

Therefore, the objective specified in **Problem 2** is equivalent to the global asymptotic stabilization of the set \mathcal{A} in (12). Next, this hybrid closed-loop system \mathcal{H}_T satisfies the hybrid basic conditions (see [23, Proposition 6.10]). The next result states that the proposed hybrid controller solves the rigid body pose tracking problem in **Problem 2**.

Theorem V.4 *The set \mathcal{A} in (12) is globally asymptotically stable for the closed-loop system \mathcal{H}_T .*

Proof: Since dynamics of the hybrid closed-loop system \mathcal{H}_T in (38) match the dynamics of the hybrid system \mathcal{H} in (11), the proof of this theorem follows the proof of Theorem V.2. \blacksquare

C. Robustness of the Closed-loop System

To be able to cope with perturbations arising in real-world settings, let us consider that the plant (10) or (38) is affected by unmodeled dynamics given by $\hat{e} = (e_1, 0) + \epsilon(e_2, 0) \in \mathcal{X}$, $e_i \in \mathbb{R}^8$, $i \in \{1, 2\}$ and measurement error $\hat{m} = (m_1, 0) + \epsilon(m_2, 0) \in \mathcal{X}$, $m_i \in \mathbb{R}^8$, $i \in \{1, 2\}$ respectively, resulting in a perturbed closed-loop system with continuous dynamics and measurements:

$$\begin{aligned} \dot{\xi} &= f(\xi) + \hat{e}, \\ y &= (\bar{q}, \bar{\omega}) + \hat{m}, \end{aligned} \quad (39)$$

where the error parameters in the original coordinates $\xi = (\bar{q}, \nu(\bar{\omega}), h) \in \widehat{\mathcal{S}}^3 \times \widehat{\mathbb{H}}^v \times \mathcal{Q} =: \mathcal{X}$ can also be defined as $\hat{e} = (\hat{e}_q, \hat{e}_\omega, 0) \in \mathcal{X}$, $\hat{e}_q := (e_{1_r} + \epsilon e_{2_t}) \in \widehat{\mathcal{S}}^3$, $\hat{e}_\omega := (e_{1_\omega} + \epsilon e_{2_v}) \in \widehat{\mathbb{H}}^v$, $\hat{m} = (\hat{m}_q, \hat{m}_\omega, 0) \in \mathcal{X}$, $\hat{m}_q := (m_{1_r} + \epsilon m_{2_t}) \in \widehat{\mathcal{S}}^3$, $\hat{m}_\omega := (m_{1_\omega} + \epsilon m_{2_v}) \in \widehat{\mathbb{H}}^v$. In addition, let us define $r := (e_1, e_2, m_1, m_2) \in \mathbb{R}^{16}$. For simplicity, the robustness results are presented only for the hybrid system (10). Note that the result in this section also holds for the hybrid system with tracking model \mathcal{H}_T in (38).

Following the fact that \mathcal{H} is well-posed and the global asymptotic stability property of the set \mathcal{A} for the

closed-loop system \mathcal{H} established in Theorem V.2, [23, Lemma 7.20] automatically leads to the following result about robustness of asymptotic stability.

Theorem V.5 *The set \mathcal{A} in (12) is semiglobally, practically robustly \mathcal{KL} asymptotically stable for the closed-loop system \mathcal{H} ; namely, there exists class- \mathcal{KL} function β such that, for each $\varepsilon > 0$ and each compact set $\mathcal{M} \subset \mathcal{X}$, there exists $\rho > 0$ such that for each measurable $r : \mathbb{R}_{\geq 0} \rightarrow \rho\mathbb{B}$, every solution ϕ to the hybrid system \mathcal{H} with initial condition $\phi(0, 0) \in \mathcal{M}$ and perturbation r satisfies*

$$|\phi(t, j)|_{\mathcal{A}} \leq \beta(|\phi(0, 0)|_{\mathcal{A}}, t + j) + \varepsilon \quad \forall (t, j) \in \text{dom } \phi. \quad (40)$$

In Theorem V.5, ‘practical’ means that the solutions to the hybrid system \mathcal{H} , in the presence of some small disturbances, converge $\varepsilon > 0$ close to the desired set \mathcal{A} in a semiglobal manner, namely, when the solutions start from arbitrary compact sets of initial conditions. A proof of this result is available in [23, Chapter 7].

VI. Simulations

A. Simulation Parameters

To verify the ideas presented in this paper we apply the hybrid hysteresis-based switching strategy to a rigid body model with mass $m = 1$ kg, and inertia

$$J = \begin{bmatrix} 1 & 0.1 & 0.15 \\ 0.1 & 0.63 & 0.05 \\ 0.15 & 0.05 & 0.85 \end{bmatrix} \text{ kg-m}^2$$

as in [34]. In the results presented below, each of the the plots show simulations of ‘hybrid’, ‘discontinuous’ and ‘continuous’, controllers. For the simulations labeled hybrid, the hysteresis half-width $\delta \in (0, 1)$ and $h(t, j) \in \{-1, 1\}$. When the hysteresis width $\delta = 0$, the controller reduces to discontinuous scheme where

$$h(t, j) := \text{sgn}(\eta_r) = \begin{cases} -1 & \eta_r < 0 \\ 1 & \eta_r \geq 0. \end{cases} \quad (41)$$

When $\delta > 1$, $h(t, j) = 1$ and a continuous controller exhibiting unwinding is implemented. To this end, simulations associated with full state feedback using hybrid feedback (7)-(9), where the output of the system (6) is measured as $y = (\hat{q}_b, \hat{\omega}_b)$ (and hence the error vector $(\bar{q}, \bar{\omega})$ is available for feedback) are presented in Section VI.B. And the simulation results associated with the hybrid tracking feedback controller (35)-(37) with measurement of dual quaternion \bar{q}_b are presented in Section V.B. Following the results in Section V.C, for all the simulation results below, the measured value of the pose $\bar{q}_m := \bar{q} + km_q/|\bar{q} + km_q|$, where $m_q = \bar{e}/|\bar{e}|$ is the normalized error. Each value of \bar{e} is drawn from a zero-mean Gaussian distribution with unit variance,

k was drawn from a uniform distribution on the interval $(0, 0.2)$ for set-point stabilization in Section VI.B and the interval $(0, 0.02)$ for the tracking control presented in Section VI.C. This additional noise in the states results in chattering behavior for the switching signal $sgn(\eta_r)$ for the discontinuous controller, while the hysteresis-based hybrid logic is impervious to such noise as shown in Figures 1-2 and Figure 6[¶].

B. Set-point Pose Stabilization

The response of the closed-loop rigid body dynamics with hybrid feedback (7)-(9) when dual quaternion error and velocity errors $(\bar{q}, \bar{\omega})$ are available for feedback is presented in Figure 1. The simulations are performed with the initial condition set to $p_b(0, 0) = (25 \text{ m}, 25 \text{ m}, 25 \text{ m})$ (position in body frame), velocity $v_b(0, 0) = (0.1 \text{ m/sec}, 0.2 \text{ m/sec}, 0.3 \text{ m/sec})$, orientation $q_{b_r}(0, 0) = (0, 0.4243, 0.5657, 0.7071)$, angular velocity $\omega_b(0, 0) = (0.2 \text{ rad/sec}, 0.4 \text{ rad/sec}, 0.6 \text{ rad/sec})$ and $h(0, 0) = 1$. The energy-based controller has the gains $k_d = 0.5$, $k_p = 0.5$ and a hysteresis gap of $\delta = 0.1$. Figure 1 also shows a comparison between the

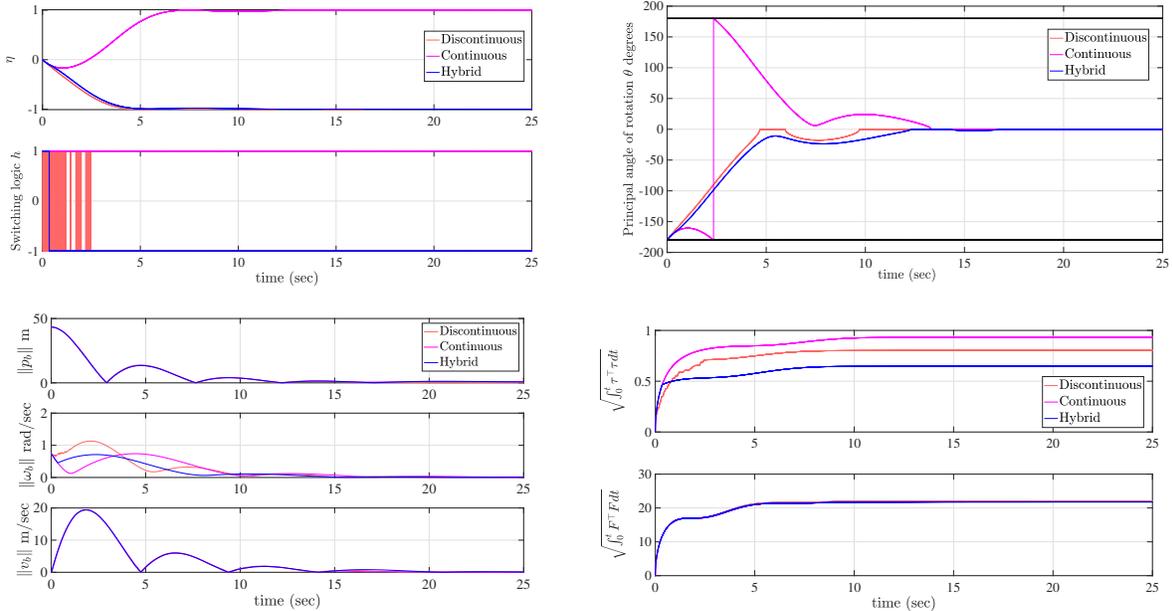


Fig. 1 Closed-loop response of the continuous, discontinuous and hybrid controllers subjected to noise with the switching logic $h(0, 0) = 1$ and $\delta = 0.1$.

linear continuous controller with $h(t, j) = 1$, a discontinuous controller where the switching logic variable $h(t, j) := sgn(\eta_r)$ as in (41) and the hybrid controller with $h(t, j) \in \{-1, 1\}$ as in Section V.A. Next, we consider a larger hysteresis width of $\delta = 0.4$ and repeat the the simulations with the same set of initial conditions and uncertainties as above. The hybrid controller now exhibits the same unwinding solution as the linear continuous controller due to the larger hysteresis gap. As discussed previously in [32], there is a correlation between hysteresis width δ and the sensitivity of the controller (7) to noise and the control effort

[¶]Code at <https://github.com/HybridSystemsLab/DualQuaternionBasedHybridController>

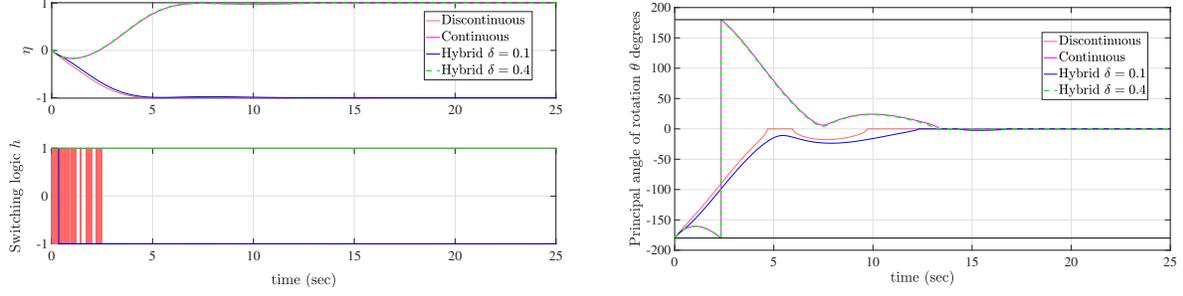


Fig. 2 Unwinding in rigid body rotational and translational dynamics with the switching logic $h(0,0) = 1$ and $\delta = 0.4$.

as shown in Figures 1- 2.

C. Pose Tracking

To simulate the rigid body pose tracking algorithm presented in Section V.B, let us consider the desired reference position and orientation satisfying Assumption V.3 are generated by the following dynamics.

$$\begin{aligned}
 \dot{q}_d &= \frac{1}{2}q_d \otimes \nu(\omega_{d/I}^d) \\
 \dot{q}_{d_t} &= \frac{1}{2}q_d \otimes \nu(v_{d/I}^d) + \frac{1}{2}q_{d_t} \otimes \nu(\omega_{d/I}^d) \\
 \dot{\omega}_{d/I}^d &= 0_{3 \times 1} \\
 \dot{v}_{d/I}^d &= (0, 0, -0.0098) - \omega_{d/I}^d \times v_{d/I}^d
 \end{aligned} \tag{42}$$

where $\omega_{d/I}^d, v_{d/I}^d$ are the angular, linear velocity of the desired frame with respect to the inertial frame expressed in the desired frame, respectively. With these set of the equations in (42), the reference pose to be tracked is generated using the following initial conditions: $q_d(0,0) = (1, 0, 0, 0)$, $q_{d_t}(0,0) = (0, 0, 0, 0)$, $\omega_{d/I}^d = (-0.1, 0.65, -0.2)$ rad/sec, $v_{d/I}^d = (-0.5, 0.1, 0.1)$ m/sec. The corresponding reference trajectory is presented in Figure 3. Next, the rigid body that tracks the reference pose in Figure 3 has the dynamics as

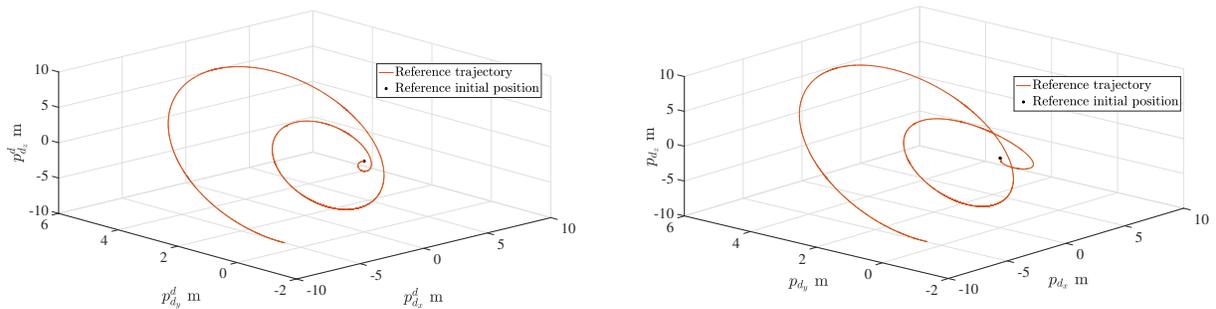


Fig. 3 Reference trajectory generated with the dynamics in (42) and resolved into the desired, body frames of reference, respectively.

given in (2) and its initial conditions are given as follows. $q_{b_r}(0, 0) = (0.1, 0.2659, 0.5318, 0.7978)$, $q_{b_t}(0, 0) = (-1.1966, -0.4318, 0.7648, -0.2159)$, $p_b = (2, 2, 1)$ m (position in body frame), $\omega_b = (-0.6, 0.6, 1)$ rad/sec, $v_b = (1, 0.5, 0.5)$ m/sec. The the hybrid feedback controller (35)-(37) is implemented with the gains $k_p = 4$, $k_d = 4$. Noise is added to the simulations as discussed in Section VI.A. As shown in Figure 4, rigid body tracks the reference orientation, position, angular and linear velocities, respectively. In addition, Figure 5 illustrates the position of the rigid body as seen in the rigid body frame of reference and desired frame of reference.

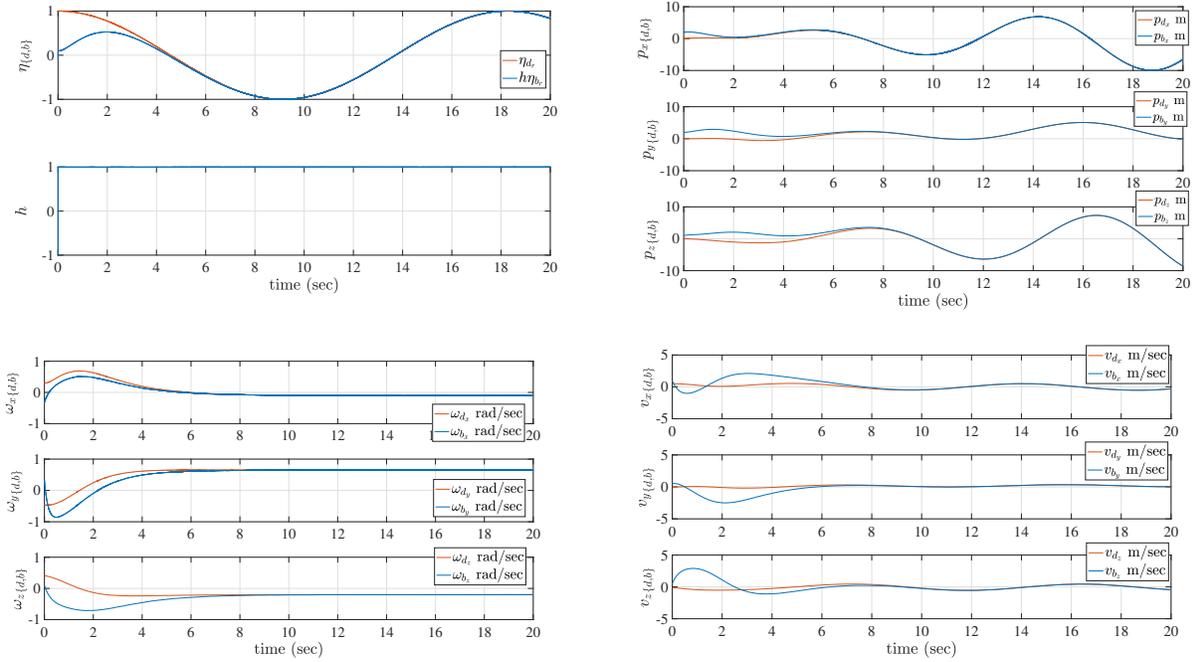


Fig. 4 Rigid body pose tracking with hybrid feedback controller (35)-(37), $h(0, 0) = 1$ and $\delta = 0.4$ resolved into the body frame of reference.

As discussed in the set-point stabilization problem in Section VI.B, a discontinuous controller where the switching logic variable $h(t, j) := \text{sgn}(\eta_r)$ as in (41) would result in chattering and not tracking the desired reference while a hybrid controller with $h(t, j) \in \{-1, 1\}$ tracks the reference pose efficiently in the presence of measurement errors. These results are illustrated in Figure 6.

VII. Conclusion

In this paper, a hybrid UDQ feedback control scheme was proposed for rigid body robust pose stabilization with full state of the system available for feedback. The stability of the closed-loop system was guaranteed through an energy-based Lyapunov function analysis using invariance principles for hybrid systems presented as set-point stabilization and tracking problems. We showed that the proposed control schemes can globally

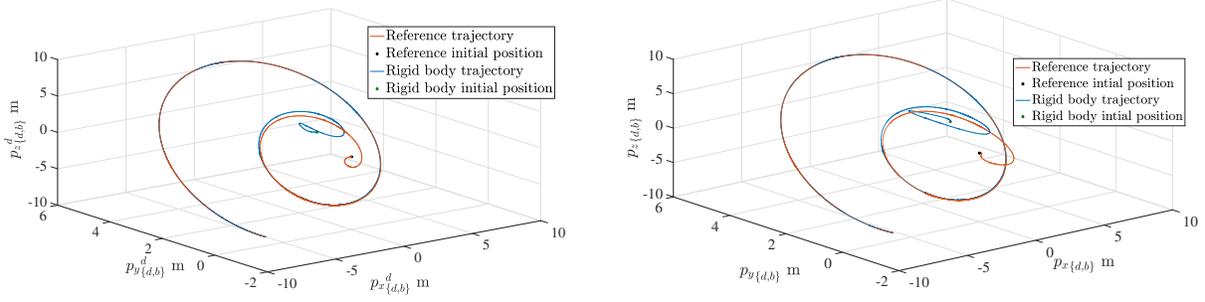


Fig. 5 Desired, rigid body trajectories expressed in desired frame of reference and rigid body frame of reference, respectively.

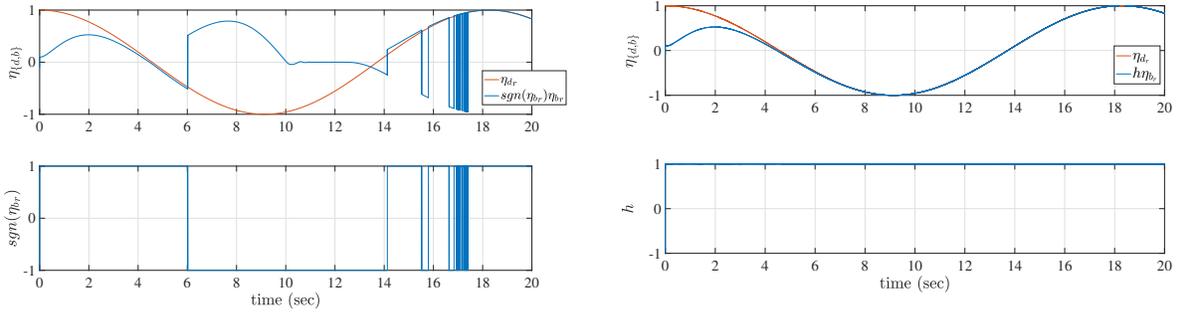


Fig. 6 Rigid body trajectories with discontinuous and hybrid controller, respectively.

asymptotically stabilize the kinematics and kinetics and establish global asymptotic stability for a rigid body. In addition, these proposed hybrid schemes allows for the controlled system to be stable in the presence of uncertainty, which would otherwise cause chattering about the point of discontinuous control. Simulation results for the rigid body motion are presented.

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VIII. Appendix

Dual Quaternions

- 1) A set of quaternions (not necessarily normalized) are denoted by $\mathbb{H} := \{q : q = (\eta, \mu), \eta \in \mathbb{R}, \mu \in \mathbb{R}^3\}$,

in which $\eta \in \mathbb{R}$ is the scalar part and $\mu \in \mathbb{R}^3$ is the vector part.

- 2) \mathcal{S}^3 denotes the set of unit quaternions, which is often used to parameterize the Lie group $\text{SO}(3)$ of rigid body attitude, where each unit quaternion is such that $|q|^2 = \eta^2 + \mu^\top \mu = 1$. Trivially, $\mathcal{S}^3 \subset \mathbb{H}$.
- 3) The set \mathcal{S}^3 has, under the quaternion product, an identity element $\mathbf{1} = (1, 0_{3 \times 1})$ and each $q = (\eta, \mu) \in \mathcal{S}^3$ has an inverse given by the quaternion conjugate $q^* = (\eta, -\mu)$.

- Note that, given $q_1, q_2 \in \mathbb{H}$, where $q_1 = (\eta_1, \mu_1)$ and $q_2 = (\eta_2, \mu_2)$, under the quaternion multiplication rule, we have:

$$q_1 \otimes q_2 = \begin{bmatrix} \eta_1 \eta_2 - \mu_1^\top \mu_2 \\ \eta_1 \mu_2 + \eta_2 \mu_1 + \mu_1 \times \mu_2 \end{bmatrix};$$

- 4) The set of dual quaternions is given by

$$\widehat{\mathbb{H}} := \{\hat{q} : \hat{q} = (\hat{\eta}, \hat{\mu}) = q_r + \epsilon q_t, \quad q_r, q_t \in \mathbb{H}\}$$

where ϵ is the unit dual, defined as $\epsilon \neq 0$, $\epsilon^2 = 0$, and given $\hat{q} = (\hat{\eta}, \hat{\mu}) \in \widehat{\mathbb{H}}$,

- $\hat{\eta} = \eta_r + \epsilon \eta_t$ is the dual scalar part of, where $\eta_r, \eta_t \in \mathbb{R}$;
- $\hat{\mu} = \mu_r + \epsilon \mu_t$ is the dual vector part, where $\mu_r, \mu_t \in \mathbb{R}^3$;
- $q_r = (\eta_r, \mu_r) \in \mathbb{H}$, where $\eta_r \in \mathbb{R}$, $\mu_r \in \mathbb{R}^3$;
- $q_t = (\eta_t, \mu_t) \in \mathbb{H}$, where $\eta_t \in \mathbb{R}$, $\mu_t \in \mathbb{R}^3$.

- 5) The space $\widehat{\mathbb{H}}^v$ denotes the dual quaternions with zero scalar part; i.e., $\widehat{\mathbb{H}}^v := \{\hat{q} = (\hat{\eta}, \hat{\mu}) \in \widehat{\mathbb{H}} : \hat{\eta} = 0\}$.
- 6) The set of dual quaternions with zero vector part is given by $\widehat{\mathbb{H}}^s := \{\hat{q} = (\hat{\eta}, \hat{\mu}) \in \widehat{\mathbb{H}} : \hat{\mu} = 0_{3 \times 1}\}$.
- 7) Given a dual quaternion $\hat{q} \in \widehat{\mathbb{H}}$, the following definitions hold:

- Conjugate: $\hat{q}^* = q_r^* + \epsilon q_t^* = (\hat{\eta}, -\hat{\mu})$;
- Swap: $\hat{q}^s = q_t + \epsilon q_r$.

where $q^* = (\eta, -\mu)$ is the conjugate of a given quaternion $q = (\eta, \mu)$.

- 8) Given any dual quaternions $\hat{q}_1, \hat{q}_2, \hat{q}_3 \in \widehat{\mathbb{H}}$, we define the following:

- a) Dual quaternion multiplication: $\hat{q}_1 \otimes \hat{q}_2 = q_{r_1} \otimes q_{r_2} + \epsilon(q_{r_1} \otimes q_{t_2} + q_{t_1} \otimes q_{r_2}) \in \widehat{\mathbb{H}}$;
- b) Dot product: $\hat{q}_1 \cdot \hat{q}_2 = \frac{1}{2}(\hat{q}_1^* \otimes \hat{q}_2 + \hat{q}_2^* \otimes \hat{q}_1) = \frac{1}{2}(\hat{q}_1 \otimes \hat{q}_2^* + \hat{q}_2 \otimes \hat{q}_1^*) = q_{r_1} \cdot q_{r_2} + \epsilon(q_{t_1} \cdot q_{r_2} + q_{r_1} \cdot q_{t_2}) \in \widehat{\mathbb{H}}^s$;
- c) Cross product: $\hat{q}_1 \times \hat{q}_2 = \frac{1}{2}(\hat{q}_1 \cdot \hat{q}_1 - \hat{q}_2^* \cdot \hat{q}_1^*) \in \widehat{\mathbb{H}}^v$;
- d) Circle product: $\hat{q}_1 \circ \hat{q}_2 = q_{r_1} \cdot q_{r_2} + q_{t_1} \cdot q_{t_2}$;
- e) Dual norm: $\|\hat{q}\|^2 = \hat{q} \otimes \hat{q}^* = \hat{q}^* \otimes \hat{q} = \hat{q} \cdot \hat{q}$
- f) $M \star \hat{q} = (M_{11}q_r + M_{12}q_t) + \epsilon(M_{21}q_r + M_{22}q_t)$, $M_{ij} \in \mathbb{R}^{4 \times 4}$, $i, j \in \{1, 2\}$.

- Note that, given a matrix $M \in \mathbb{R}^{4 \times 4}$ and a quaternion $q = (\eta, \mu) \in \mathbb{H}$,

$$Mq = (m_{11}\eta + m_{12}\mu, m_{21}\eta + m_{22}\mu) \in \mathbb{H},$$

where $m_{11} \in \mathbb{R}$, $m_{12} \in \mathbb{R}^{1 \times 3}$, $m_{21} \in \mathbb{R}^{3 \times 1}$, $m_{22} \in \mathbb{R}^{3 \times 3}$ are entries of

$$M = \begin{bmatrix} m_{11} & m_{12} \\ m_{21} & m_{22} \end{bmatrix}.$$

9) The zero dual quaternion is given by $\hat{\mathbf{0}} = \mathbf{0} + \epsilon\mathbf{0}$.

10) The dual quaternions $\hat{q}_1, \hat{q}_2, \hat{q}_3 \in \widehat{\mathbb{H}}$ satisfy the following properties.

a) $\hat{q}_1 \circ (\hat{q}_2 \otimes \hat{q}_3) = \hat{q}_2^s \circ (\hat{q}_1^s \otimes \hat{q}_3^s) = \hat{q}_3^s \circ (\hat{q}_2^s \otimes \hat{q}_1^s);$

b) $\hat{q}_1 \circ (\hat{q}_2 \times \hat{q}_3) = \hat{q}_2^s \circ (\hat{q}_3 \times \hat{q}_1^s) = \hat{q}_3^s \circ (\hat{q}_1^s \times \hat{q}_2);$

c) $(M \star \hat{q}_1) \circ \hat{q}_2 = \hat{q}_1 \circ (M^T \star \hat{q}_2);$

d) $\hat{q}_1 \circ \hat{q}_2 = \hat{q}_2 \circ \hat{q}_1;$

e) $\hat{q}_1^s \circ \hat{q}_2^s = \hat{q}_1 \circ \hat{q}_2;$

f) $(\hat{q}_1^s)^s = \hat{q}_1;$

g) $\hat{q}_1 \times \hat{q}_1 = \hat{\mathbf{0}}.$

11) The set of unit dual quaternions is denoted by $\widehat{\mathcal{S}}^3$, where each unit dual quaternion $\hat{q} = q_r + \epsilon q_t \in \widehat{\mathbb{H}}$, $q_r, q_t \in \mathbb{H}$, under the dual norm

$$\|\hat{q}\|^2 = \hat{q} \otimes \hat{q}^* = \hat{q}^* \otimes \hat{q} = q_r \otimes q_r^* + \epsilon(q_r \otimes q_t^* + q_t \otimes q_r^*),$$

is such that $q_r \otimes q_r^* = \mathbf{1}$ and $q_r \otimes q_t^* + q_t \otimes q_r^* = \mathbf{0}$

12) The set $\widehat{\mathcal{S}}^3$ has, under the dual quaternion multiplication, an identity element $\hat{\mathbf{1}}$, where $\hat{\mathbf{1}} = \mathbf{1} + \epsilon\mathbf{0}$, $\mathbf{1} = (1, 0_{3 \times 1})$, $\mathbf{0} = (0, 0_{3 \times 1})$ and the inverse given by the dual quaternion conjugate \hat{q}^* .

13) Given an invertible matrix $M \in \mathbb{R}^{n \times n}$, we define the operation $M \star M^{-1} := I_n$.

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